| Page | Place | Error | It should be |
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| 57 | equation 6.4 | $k_{\text {large_angle }}=\frac{K G}{L}+\frac{1}{120} E\left(\frac{\varphi^{2}}{L^{3}}\right) t b^{5}$ | $k_{r_{\text {large_angle }}}=\frac{K G}{L}+\frac{1}{360} E\left(\frac{\varphi^{2}}{L^{3}}\right) t b^{5}$ |
| 57 | below equation 6.4 | ... the factor of stiffness $k_{\text {large_angle }}$ divided by the torsional stiffness $k_{\mathrm{I}}$ is plotted against the angle to length ratio $\theta / L \ldots$ | ... the factor of stiffness $k$ $\qquad$ divided by the torsional stiffness $k_{\mathrm{I}}$ is plotted against the angle to length ratio $\varphi / L \ldots$ |
| 58 | figure 6.7 | figure of stiffening should be as in figure: |  |
| 60 | calculation | $C_{\mathrm{w}}=\ldots . .=8.10 \cdot 10^{5} \mathrm{~mm}^{4}$ | $C_{\mathrm{w}}=\ldots . .=8.10 \cdot 10^{5} \mathrm{~mm}^{6}$ |
| 61 | calculation | $\beta=\ldots . .=3.04 \cdot 10^{3} \frac{1}{\mathrm{~mm}}$ | $\beta=\ldots . .=3.04 \cdot 10^{-3} \frac{1}{\mathrm{~mm}}$ |
| 61 | calculation | $k_{\mathrm{rI}_{\mathrm{I}}}=\ldots . .=15.7 \frac{\mathrm{Nmm}}{\mathrm{rad}}$ | $k_{\mathrm{r}_{\mathrm{I}}}=\ldots . . .=1.57 \cdot 10^{4} \frac{\mathrm{Nmm}}{\mathrm{rad}}$ |
| 61 | calculation | $k_{\mathrm{rIII}}=\ldots . .=529 \frac{\mathrm{Nmm}}{\mathrm{rad}}$ | $k_{\text {rII }}=\ldots . .=5.29 \cdot 10^{5} \frac{\mathrm{Nmm}}{\mathrm{rad}}$ |


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| 61 | calculation | $k_{\mathrm{r}_{\mathrm{III}}}=\ldots . .=20.6 \cdot 10^{2} \frac{\mathrm{Nmm}}{\mathrm{rad}}$ | $k_{\mathrm{rIII}}=\ldots .=2.06 \cdot 10^{6} \frac{\mathrm{Nmm}}{\mathrm{rad}}$ |
| 61 | calculation | $k_{\mathrm{rIV}}=\ldots . .=40.5 \cdot 10^{2} \frac{\mathrm{Nmm}}{\mathrm{rad}}$ | $k_{\mathrm{rIV}}=\ldots .=4.05 \cdot 10^{6} \frac{\mathrm{Nmm}}{\mathrm{rad}}$ |
| 66 | equation 6.16 line 1 | $\sigma_{\max }=\ldots . \quad \text { for } \quad a \geq \frac{1}{2}$ | $\sigma_{\max }=\ldots . \quad$ for $\quad a \leq \frac{1}{2}$ |
| 66 | $\begin{array}{ll} \text { equation } & 6.16 \\ \text { line } 3 & \end{array}$ | $\sigma_{\max }=\ldots . \quad \text { for } \quad a \leq \frac{1}{2}$ | $\sigma_{\max }=\ldots . \quad \text { for } \quad a \geq \frac{1}{2}$ |
| 77 | figure 6.34e | lenght $p$ | lenght $p L$ |
| 90 | equation 8.7 | $\delta_{\mathrm{sh}}=-\frac{1}{2} \frac{\delta_{\mathrm{c}}^{2}}{L}$ | $\delta_{\text {sh }}=-\frac{1}{2} \frac{\delta^{2}}{L}$ |
| 90 | last paragraph | The shortening of the simple beam is easier to derive when it is modelled as two cantilever beams with a force at the ends. | This is valid for simple beams where the force is applied exactly in the middle (and an approximation for forces close to the middle). |
| 122 | first line below phase angle | reference to equation 10.14 | should be to equation 10.15 |
| 164 | equation 13.3 | $k_{A C}$ | $k_{B C}$ |


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| 168 | thickness in figure of thin tube | $t=q$ | $t=\frac{q}{2}$ |
| $\begin{aligned} & 170 \\ & 171 \end{aligned}$ | equations 13.12, 13.13, 13.15 | $\theta$ could have been expressed in $L$ and $h$ | $k_{\text {truss }}=\frac{E h^{3} L t}{4 L^{4}+4 L^{2} h^{2}+h^{4}}$ |
| 171 | equations 13.14, 13.16 | $h t^{3}$ | $h^{3} t$ |
| 171 | $\begin{array}{ll}\text { above } & \text { figure } \\ 13.27 & \end{array}$ | height to length ratio $h / L$ | length to height ratio $L / h$ |
| 171 | last sentence | stiffness almost the same as the stiffness of the sheet | that is not the case, it is not almost the same |
| 172 | figure 13.28 | stiffness of symmetrical truss should be as in figure: |  |
| 172 | equation 13.18 | equation of stiffness is wrong | $k_{\text {truss2 }}=\frac{4 E L h^{3} t}{16 L^{4}+8 L^{2} h^{2}+h^{4}}$ |
| 172 | below equation 13.18 | stiffness almost the same as the stiffness of the sheet | that is not the case, it is not almost the same |


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| 178 | equation 13.24 | $\omega_{1}=\ldots$ | $\omega_{n 1}=\ldots \ldots$ |
| 190 | equation 13.31 | $L(1-\cos \alpha) \approx \frac{\alpha^{2}}{2}$ | $L(1-\cos \alpha) \approx \frac{\alpha^{2} L}{2}$ |
| 224 | equation 16.4 | $U=\ldots .=\frac{1}{2} m \omega^{2} r^{2}(\sin (\omega t)+\cos (\omega t))^{2}=m \omega^{2} r^{2}$ | $U=\ldots .=\frac{1}{2} m \omega^{2} r^{2}\left(\sin (\omega t)^{2}+\cos (\omega t)^{2}\right)=\frac{1}{2} m \omega^{2} r^{2}$ |
| 225 | equation 16.4 | $y=L_{\text {beam }} \cos (\theta) z=L_{\text {beam }} \sin (\theta)$ | $y=L_{\text {beam }} \cos (\theta) \quad z=L_{\text {beam }} \sin (\theta)$ |
| 226 | figure 16.3 | $h_{0}$ | $L_{0}$ |
| 226 | equation 16.8, line 3 | $k L_{A} L_{B} \sin (\theta)+m g L_{C} \sin (\theta)=0$ | $k L_{A} L_{B} \sin (\theta)-m g L_{C} \sin (\theta)=0$ |
| 277 | table C. 2 , line 3 | $k=\frac{3 E I}{L^{3} a^{2}\left(a-2 a+a^{2}\right)}$ | $k=\frac{3 E I}{L^{3} a^{2}\left(1-2 a+a^{2}\right)}$ |
| 280 | table C.5, line 8 | $K=b t^{3}\left(\frac{1}{3}-0.21\left(1-\frac{t^{4}}{12 b^{4}}\right)\right)$ | $K=b t^{3}\left(\frac{1}{3}-0.21 \frac{t}{b}\left(1-\frac{t^{4}}{12 b^{4}}\right)\right)$ |

