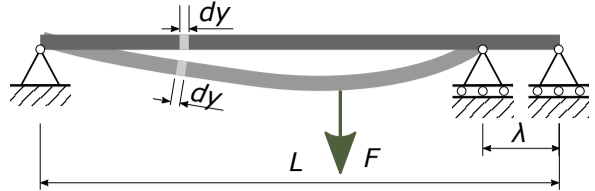
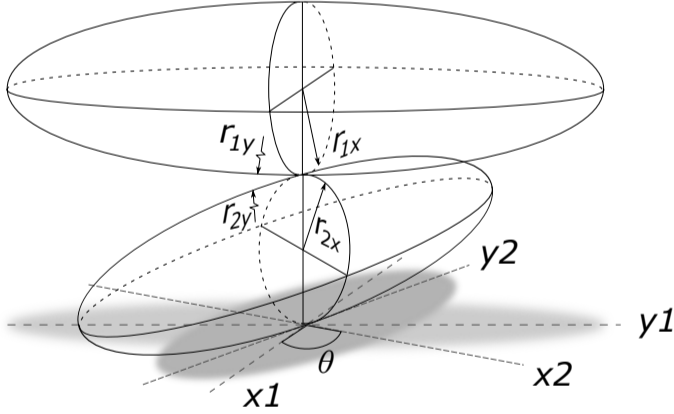


| Page | Place | Error | It should be |
|------|-----------------------------|--|--|
| 11 | figure 2.52 | In the left figure the length of the partition is called $d\lambda$ |  |
| 14 | first line of 2.5 | between to | between two (or more) |
| 28 | formula 2.28 | $\int_{F_1}^{F_2} \frac{dF}{F} = \int_0^\varphi \mu d\theta = \ln \frac{F_2}{F_1} = \mu\varphi \Rightarrow \frac{F_2}{F_1} = e^{\mu\varphi}$ | $\int_{F_2}^{F_1} \frac{dF}{F} = \int_0^\varphi \mu d\theta = \ln \frac{F_1}{F_2} = \mu\varphi \Rightarrow \frac{F_1}{F_2} = e^{\mu\varphi}$ |
| 40 | below formula 4.9 | If i is much smaller than 1 (if b becomes much bigger than a) | If i is much greater than 1 (if a becomes much greater than b) |
| 42 | below formula 4.15 | If i is much smaller than 1 (if r_A becomes much bigger than r_B) | If i is much greater than 1 (if r_B becomes much greater than r_A) |
| 45 | formula 4.28 and line below | $U = \frac{1}{2} \frac{F_{in}^2}{k_{sys}} = \frac{1}{2} \frac{F_{AC}^2}{k_{AC}} + \frac{1}{2} \frac{F_{BC}^2}{k_{BC}} = \frac{1}{2} \frac{\left(F_{in} \cdot \frac{\sin(\beta)}{\sin(\theta)}\right)^2}{k_{AC}} + \frac{1}{2} \frac{\left(F_{in} \cdot \frac{\sin(\alpha)}{\sin(\theta)}\right)^2}{k_{BC}}$ The stiffness of the truss can be derived by dividing the energy formula by $\frac{1}{2}F_{in}^2$. | $U = \frac{1}{2} \frac{F_{ext}^2}{k_{sys}} = \frac{1}{2} \frac{F_{AC}^2}{k_{AC}} + \frac{1}{2} \frac{F_{BC}^2}{k_{BC}} = \frac{1}{2} \frac{\left(F_{ext} \cdot \frac{\sin(\beta)}{\sin(\theta)}\right)^2}{k_{AC}} + \frac{1}{2} \frac{\left(F_{ext} \cdot \frac{\sin(\alpha)}{\sin(\theta)}\right)^2}{k_{BC}}$ The stiffness of the truss can be derived by dividing the energy formula by $\frac{1}{2}F_{ext}^2$. |
| 48 | line below formula 4.38 | Dividing formula 4.38 by $\frac{1}{2}F_{in}^2$ | Dividing formula 4.38 by $\frac{1}{2}F^2$ |
| 50 | line 4 in 5.1.3 | bending (a) and shear (b) | shear (a) and bending (b) |
| 51 | caption of figure 5.3 | Deformation due to a) bending and b) shear | Deformation due to a) shear and b) bending |
| 51 | formula 5.4 | $k_{shear} = \frac{G \cdot A}{f_s}$ | $k_{shear} = \frac{G \cdot A}{f_s \cdot L}$ |
| 52 | table 5.1 | Form factor rectangle $\frac{5}{6}$ | Form factor rectangle $\frac{6}{5}$ |
| 56 | table 5.2 top 3 rows | 0, 0, $\neq 0$ | The C_w is not applicable for closed profiles. If there would be a value it would probably rather be very high than zero. |
| 65 | figure 6.1, vertical axis | F | k |
| 68 | figure 6.5 | lines for r_{1y} and r_{2y} are drawn from the midpoints of the ellipsoids |  |
| 71 | formula 6.15 | $\delta_A = \frac{F}{L} \frac{1 - \nu_A^2}{\pi E_A} \left(2 \ln \left(\frac{2d_A}{a_A} \right) - \frac{\nu_A}{1 - \nu_A} \right)$ | $\delta_A = \frac{F}{L} \frac{1 - \nu_A^2}{\pi E_A} \left(2 \ln \left(\frac{2d_A}{a_{AB}} \right) - \frac{\nu_A}{1 - \nu_A} \right) \quad (\text{approximation})$ or $\delta_A = \frac{F}{L} \frac{1 - \nu_A^2}{\pi E_A} \left(2 \ln \left(\frac{4d_A}{a_{AB}} \right) - 1 \right) \quad \text{for } d_A < R_A$ |
| 71 | formula 6.16 and 6.18 | $a_A = \sqrt{\frac{4FR_A}{L\pi E^*}} \quad \text{and} \quad a_B = \sqrt{\frac{4FR_B}{L\pi E^*}}$ | $a_{AB} = \sqrt{\frac{4FR_c}{L\pi E^*}} \quad \text{with} \quad R_c = \left(\frac{1}{r_A} + \frac{1}{r_B} \right)^{-1}$ |
| 71 | formula 6.17 | $\delta_B = \frac{F}{L} \frac{1 - \nu_B^2}{\pi E_B} \left(2 \ln \left(\frac{2d_B}{a_B} \right) - \frac{\nu_B}{1 - \nu_B} \right)$ | $\delta_B = \frac{F}{L} \frac{1 - \nu_B^2}{\pi E_B} \left(2 \ln \left(\frac{2d_B}{a_{AB}} \right) - \frac{\nu_B}{1 - \nu_B} \right) \quad (\text{approximation})$ or $\delta_B = \frac{F}{L} \frac{1 - \nu_B^2}{\pi E_B} \left(2 \ln \left(\frac{4d_B}{a_{AB}} \right) - 1 \right) \quad \text{for } d_B < R_B$ |